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Simultaneous estimation of mass and aerodynamic rotor imbalances for wind turbines

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Abstract

The safe operation of wind turbines requires a well-balanced rotor. The balancing of the rotor requires a method to determine its imbalances. We propose an algorithm for the reconstruction of two types of imbalances, i.e., mass and aerodynamic imbalances from pitch angle deviation. The algorithm is based on the inversion of the (nonlinear) operator equation that links the imbalance distribution of the rotor to its vibrations during operation of the wind turbine. The algorithm requires a simple finite element model of the wind turbine as well as the minimization of a Tikhonov functional with a nonlinear operator. We propose the use of a gradient-based minimization routine. The approach is validated for artificial vibration data from a model of a Nordwind NTK 500 wind turbine.

Keywords: rotor imbalance; inverse problem; nonlinear Tikhonov regularization

1 Introduction

In the growing field of wind energy extraction, the topic of rotor imbalances of wind turbines (WT) is of vital importance for the operation, safety, and durability of the turbines. Approximately 20%-50% of WT have significant rotor imbalances. This leads to damages of important components, high repair expenses, and reduced output [1]. Rotor imbalances can be categorized into mass and aerodynamic imbalances. Mass imbalances arise from inhomogeneous mass distributions of the rotor including blades and hub. They induce centrifugal forces that depend on the square of the rotational frequency Ω_{rot} . The consequences are harmonic vibrations with Ω_{rot} in the rotor plane (lateral). The vibration amplitude depends on the ratio between Ω_{rot} and the resonance frequency (first eigenfrequency) Ω_0 of the turbine. Mass imbalances can be eliminated or reduced using balancing counterweights. Aerodynamic imbalances evolve from deviations in the aerodynamic properties of the blades. These lead to different thrust and tangential forces and moments, and they depend not only on the rotational frequency but also on the wind speed. Aside from lateral vibrations, they induce axial and torsional vibrations of WT. The main reason for aerodynamic imbalance is a relative deviation of the pitch angles of the blades. This can only be eliminated by a correction of those angles. Additional masses are not helpful here [1]. In practice, both types of imbalances are frequently observed both separately and in combination.

Usually, the detection of imbalances is based on spectral analysis and order analysis methods [2]. Another technique is to monitor the power characteristic [3], whereby power mean values are observed, and deviations from faultless conditions are used for the calculation of alarm limits. Unfortunately, this requires a learning phase under faultless conditions. This also holds for other signal processing methods, cf. [2]. Another disadvantage is the fact that the amount of the imbalance, e.g. absolute value and location of a mass imbalance or the deviation of one or more pitch angles, is not computable.

The state of the art in rotor imbalance determination still requires an expert team on-site. Initially, aerodynamic imbalances must be detected using optical methods. After the elimination of the aerodynamic imbalances, the mass imbalance is determined using vibration measurements with and without reference weights.

In the last few years, the authors have developed methods which allow for imbalance determination using data that can be collected by a condition monitoring system (CMS). As a consequence, the amount and location of rotor imbalance can be computed without the expensive measurements using reference weights on-site. The methods are model-based, but they use a very simple model of the WT that can be obtained easily for each turbine type from elementary geometrical and physical parameters. The general basics of our approach are summarized in Section 2. Essentially, the method consists of the following steps: 1. modeling of the WT; 2. description of the loads (forces, moments) arising from rotor imbalance; 3. solution of the forward problem, i.e. the computation of the resulting vibrations in every node of the WT model for a given imbalance situation and 4. solution of the inverse problem where the unknown imbalance is determined from vibration data obtained in the nacelle. In our new method we began by determining the mass imbalance [4]. Later aerodynamic imbalance from pitch angle deviations was included [5].

In case of pure mass imbalances, only lateral vibrations need to be considered. Accordingly, in the model of the WT only degrees of freedom (DOF) in the lateral direction needed to be taken into account. Let us call this restricted approach the mass imbalance approach (MIA). The presence of aerodynamic imbalances in addition to mass imbalances would lead to additional lateral forces and therefore to incorrect reconstructions using MIA since the assumption in this model is that aerodynamic imbalances are not present. The model and the description of the forces were expanded in [5] in order to include the effect of pitch angle deviations as a main reason for aerodynamic imbalances. Nevertheless, MIA can serve to provide an initial guess for the solution of the problem for combined mass and aerodynamic imbalances. In the combined imbalance case, the inverse problem is nonlinear. As the problem is ill-posed, Tikhonov regularization is used. This includes the minimization of the Tikhonov functional which was done in [5] using a direct search method. Although we could obtain first test results, the routine was far too slow and not stable enough for extensive test computations. The acceleration of the minimization routines requires the computation of the Frechet derivative of the forward problem operator \mathbf{A} . As most of the theory for Tikhonov regularization is formulated for real valued vector spaces, the forward operator \mathbf{A} has to be formulated in a real setting.

This paper focuses on a stable reconstruction method of mass imbalance and pitch angle deviation based on the minimization of the Tikhonov functional. To this end, the nonlinear forward operator \mathbf{A} is derived in detail and its Frechet derivative is computed which is new compared to [5]. Now the minimization of the Tikhonov functional is performed using the steepest decent method. The resulting reconstruction algorithm is stable and

reasonable fast. Additionally, a suitable metric is defined to measure the reconstruction quality. We present test results obtained with the gradient-based algorithm for artificial data. The paper is organized as follows: In Section 3 we derive the forward problem operator \mathbf{A} . We will be very brief if there are no changes to the preceding paper [5] and more detailed in the description of the new parts. In particular, the Frechet derivative of \mathbf{A} is computed. Section 4 is concerned with the inverse problem. Regularization and minimization techniques using gradient-based methods are presented as well as the final algorithm that reconstructs the imbalances from measurement data. In Section 5 we present numerical test computations for the reconstruction of pitch angle deviations only and a combination of mass imbalances and pitch angle deviations as well as results for the reconstruction quality in the presented metric.

2 General approach

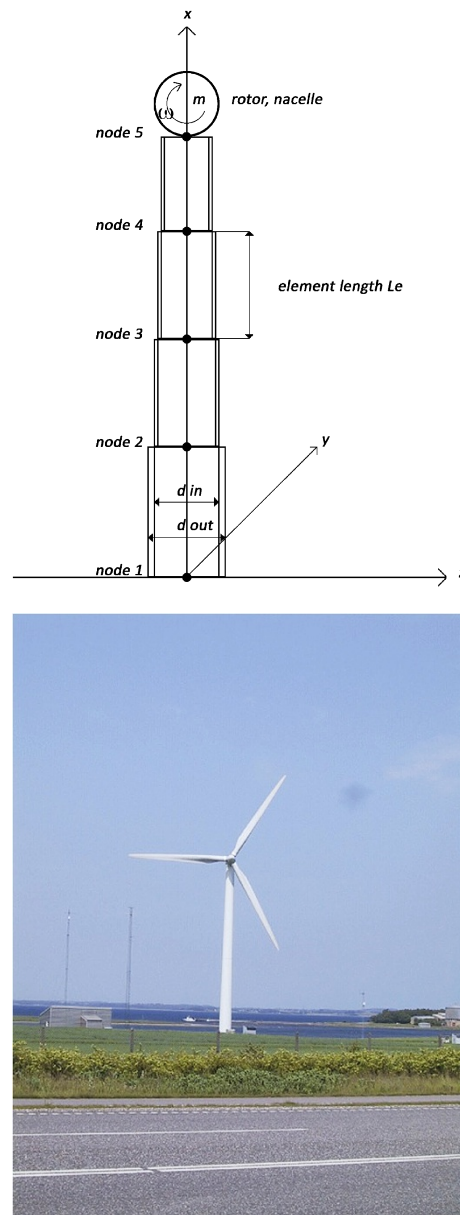
The procedure to determine imbalances based on a model of the WT can be summarized as follows: The load $p(\mathbf{x})$ arising from an imbalance \mathbf{x} induces displacements or vibrations u of the WT in different directions. Here we deal with displacements in axial direction (along the y -axis), lateral displacements (along the z -axis), and torsion around the tower center axis (x -axis), cf. Figure 1. The displacements and the imbalance loads are coupled via a Partial Differential Equation (PDE) in time and space which is seldom explicitly solvable. Therefore, we use the Finite Element Method (FEM) to transform the PDE into a system of Ordinary Differential Equations (ODE) in time. To this end we consider the rotor and nacelle as point masses and the WT tower as a flexible shaft that is divided into n elements with $n + 1$ nodes (see Figure 1). The ODE system reads

$$\mathbf{M}\ddot{\mathbf{u}}(t) + \mathbf{S}\mathbf{u}(t) = \mathbf{p}(t), \quad (1)$$

where \mathbf{M} and \mathbf{S} represent the mass and stiffness matrices, \mathbf{u} the vector of displacements or degrees of freedom (DOF) of each node, and \mathbf{p} the vector of loads for each DOF. There could be an additional damping term $\mathbf{D}\dot{\mathbf{u}}(t)$ on the left hand side of (1), but so far we have achieved sufficiently good results with damping neglected.

The matrices \mathbf{M} and \mathbf{S} need to be derived for each different type of WT separately. This is the *first step* in our procedure. The *second step* pertains to the description of the load $p(\mathbf{x})$ as a function of the imbalances \mathbf{x} . In the *third step*, the now fully described ODE system (1) is solved. The solution operator $\tilde{\mathbf{A}}$ maps the imbalances \mathbf{x} to the displacement \mathbf{u} at each node of the FE model: $\tilde{\mathbf{A}}\mathbf{x} = \mathbf{u}$. Now we restrict the displacement vector \mathbf{u} that contains the displacement of each model DOF to the vector \mathbf{y} of the DOF that are accessible for measurements. The operator $\tilde{\mathbf{A}}$ is accordingly restricted to \mathbf{A} . The resulting operator equation is $\mathbf{A}\mathbf{x} = \mathbf{y}$ where the operator \mathbf{A} maps the imbalance \mathbf{x} of the rotor in terms of mass imbalance and pitch angle deviation to the displacement \mathbf{y} in the nodes that can be observed by measurements. This operator equation represents the forward problem. Hence *step 1-3* are connected with the solution of the forward problem. Now in a *fourth step* we solve the inverse problem, i.e., we determine the imbalance \mathbf{x} from measurements \mathbf{y} . Since the inverse problem is ill-posed, we use regularization techniques, [6]. This approach has been applied successfully in the more simple case of mass imbalances only [4]. First steps in the direction of reconstructing mass imbalances and pitch angle deviation simultaneously were done in [5].

Figure 1 Model of a wind turbine and real wind turbine. Top: Finite-Element model of a WT with coordinate system. Bottom: NTK500/41 in Denmark.



3 Forward problem

3.1 Mass and stiffness matrices

The derivation of \mathbf{M} and \mathbf{S} for the simple model of MIA was taken from [7] and presented in [4]. In the situation of mass and aerodynamic imbalances the model needs to be expanded to displacements in axial direction and torsion around the tower central axis. Still, the derivation of \mathbf{M} and \mathbf{S} follows exactly the Ritz method described in [7, p.155-160]. Once the model matrices are derived for a certain WT type they need to be optimized for each individual turbine of this type with respect to the first eigenfrequency Ω_0 . The general optimization procedure is described in [4].

Thus far we have modeled six different types of WT in this way: Vestas V80, V82, V90, Südwind S77, NTK500/41 and GE1.5. For the test computations presented in this paper,

we have used the model of a Nordtank turbine NTK500/41, see Figure 1 (bottom), with 35 m hub height because we are provided with aerodynamic data for the blades of this WT.

3.2 Forces and moments from imbalances – the load vector \mathbf{p}

The imbalance of the rotor creates forces and moments that affect the top node of the FE model. The forces and moments are ordered in accordance to the DOF in the displacement vector \mathbf{u} :

$$\mathbf{p}^T(t) = (0, \dots, 0, F_y, F_z, M_x, M_y, M_z), \quad (2)$$

where F_y and F_z denote forces in y - and z -direction, M_x , M_y , and M_z are moments of forces or torques w.r.t. the x -, y - and z -axis, respectively. In particular, we have $M_y = 0$ as we assume that mechanical energy related to M_y is transformed into electrical energy. The forces and moments are the sum of the forces and moments from mass imbalance and pitch angle deviation:

$$\begin{aligned} F_y &= F_1 + F_2 + F_3, \\ F_z &= T_z + (F_c)_z, \\ M_x &= M_x^{(m)} + M_x^{(F)} + M_x^{(T)}, \\ M_z &= M_z^{(m)} + M_z^{(F)} + M_z^{(T)}. \end{aligned} \quad (3)$$

Here, the F_1, F_2, F_3 are the thrust forces at each blade. If there is no deviation in the pitch angle of the blades all three forces are equal. T_z is the projection of the sum of the tangential forces at each blade $T = T_1 + T_2 + T_3$ onto the z -axis. It is zero if there is no deviation in the pitch angles. F_c denotes the centrifugal force induced by the mass imbalance. Its projection onto the z -axis $(F_c)_z$ adds to F_z . The moments $M_x^{(m)}, M_z^{(m)}$ are induced by F_c of the mass imbalance, $M_x^{(F)}, M_z^{(F)}, M_x^{(T)}, M_z^{(T)}$ correspond to the thrust force and tangential force. Whereas the mass imbalance force can be directly computed, the aerodynamic imbalance forces need to be determined via, e.g., the Blade-Element-Momentum (BEM) method, a nonlinear procedure. In the BEM method, the blades are divided into a finite number of elements which are annulus segments with the center at the root of the blades. The cross section of each element is called an 'airfoil'. If the airfoil data, the angle of attack of the wind, and the relative wind velocity as well as a lift and drag coefficient table are given, the normal or thrust forces F and the tangential forces T can be calculated according to the BEM method, see, e.g., [8]. The BEM method is used in a lot of programs for the simulation of aerodynamic effects, cf. [9], which were tested on real WT.

For the presentation of the forces and moments, we will use the following notations and abbreviations (cf. Figures 2 and 3):

- mr, ϕ_m – absolute value and angle w.r.t. blade A of the mass imbalance,
- θ_i – deviation of the pitch angle of i th blade to an adjusted pitch angle θ ,
- $\omega = 2\pi\Omega$ – angular frequency (note: WT rotates clockwise seen from the wind),
- $\varphi = 2/3\pi$ – angle between blades,
- ϕ – initial location of blade A w.r.t. a zero mark (equals 0 if the mark can be chosen at blade A),
- L – distance from hub to nacelle midpoint (x -axis),

Figure 2 Notations and force from mass imbalance.

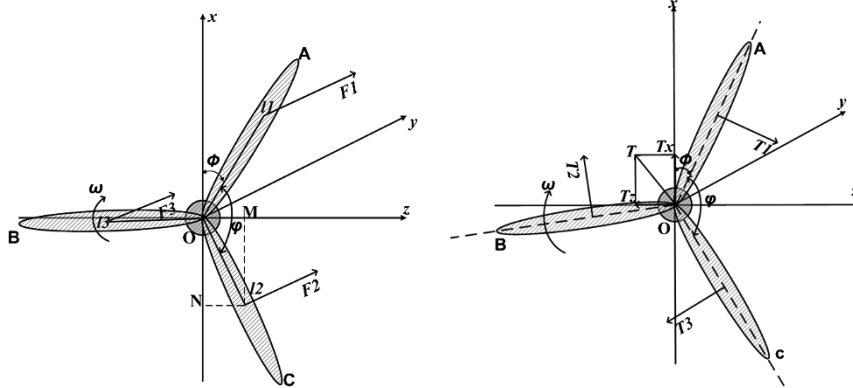
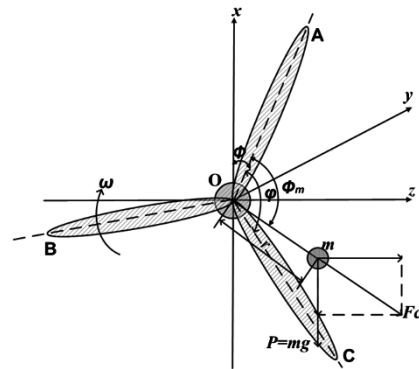


Figure 3 Thrust and tangential forces. Left: Thrust forces F_1, F_2, F_3 and corresponding lengths. Right: Tangential forces T_1, T_2, T_3 .

- $F_i := F(x_i)$ – thrust force at blade i with pitch angle deviation θ_i , $i = 1, 2, 3$,
- $T_i := T(x_i)$ – tangential force at blade i with pitch angle deviation θ_i , $i = 1, 2, 3$,
- $l_i := l(x_i)$ – corresponding length (distance of resulting force vector from nacelle) for pitch angle θ_i , $i = 1, 2, 3$.

3.2.1 Forces and moments from mass imbalance

The centrifugal force of a mass m with the distance r from the hub rotating with the frequency ω has the absolute value

$$|F_c| = \omega^2 mr.$$

The projections of F_c onto the z - and x -axis are

$$\begin{aligned} (F_c)_z &= |F_c| \sin(\omega t + \phi + \phi_m), \\ (F_c)_x &= |F_c| \cos(\omega t + \phi + \phi_m), \end{aligned} \quad (4)$$

cf. Figure 2. The rotor has a small distance L to the tower center, the x -axis. Thus the forces induce moments given by

$$M_x^{(m)} = (F_c)_z \cdot L, \quad M_z^{(m)} = (F_c)_x \cdot L. \quad (5)$$

3.2.2 Forces and moments from pitch angle deviation

The thrust force F and the tangential force T on a blade depend on the geometry and aerodynamics of the blade, the wind speed, the rotational speed, and the pitch angle. If the pitch angles are not the same for every blade the forces are also different. The derivation of the thrust forces F_1, F_2, F_3 , the tangential forces T_1, T_2, T_3 and the corresponding lever arms l_1, l_2, l_3 (cf. Figure 3) uses the nonlinear Blade Element Momentum (BEM) method. This is described in detail in [5, 8]. The BEM method is used to compute the forces for each blade element. For our purpose the distributed forces are converted into one equivalent force at a corresponding lever arm l_i , cf. Figure 3. Also from Figure 3 we observe that the thrust forces F_i act in y direction. Thus

$$F_y = F_1 + F_2 + F_3. \quad (6)$$

The forces create moments around the x - and the z -axis:

$$\begin{aligned} M_x^{(F)} &= F_1 l_1 \sin(\omega t + \phi) + F_2 l_2 \sin(\omega t + \phi + \varphi) + F_3 l_3 \sin(\omega t + \phi + 2\varphi), \\ M_z^{(F)} &= F_1 l_1 \cos(\omega t + \phi) + F_2 l_2 \cos(\omega t + \phi + \varphi) + F_3 l_3 \cos(\omega t + \phi + 2\varphi). \end{aligned} \quad (7)$$

The total tangential force

$$T = T_1 + T_2 + T_3$$

has components in x - and z -direction:

$$\begin{aligned} T_z &= T_1 \sin(\omega t + \phi) + T_2 \sin(\omega t + \phi + \varphi) + T_3 \sin(\omega t + \phi + 2\varphi), \\ T_x &= T_1 \cos(\omega t + \phi) + T_2 \cos(\omega t + \phi + \varphi) + T_3 \cos(\omega t + \phi + 2\varphi). \end{aligned} \quad (8)$$

As mentioned before we have a small distance L between rotor plane and tower center, thus T_z and T_x also induce moments around the x - and the z -axes:

$$M_x^{(T)} = T_z \cdot L, \quad M_z^{(T)} = T_x \cdot L. \quad (9)$$

3.2.3 Combination and rewriting of forces and moments – the final load vector

We observe that F_y is the only constant force, whereas all other forces and moments are harmonic with the rotating angular frequency ω . The system's answer to a constant force are vibrations with the natural eigenfrequencies $\omega_0, \omega_1, \dots$ of the WT centered around a constant displacement:

$$\mathbf{u}_1(t) = [\mathbf{I} - \cos(\sqrt{\mathbf{M}^{-1}\mathbf{S}}t)]\mathbf{S}^{-1}\mathbf{p}_1(\mathbf{x}),$$

where $\mathbf{p}_1^T := (0, \dots, 0, F_y, 0, 0, 0, 0)$, and the ω_i are the roots of the eigenvalues of $\mathbf{M}^{-1}\mathbf{S}$. For WT the only eigenfrequency close to the operating frequency ω is ω_0 . For safety reasons, the operation of the WT with $\omega = \omega_0$ is avoided. As only vibrations with rotating frequency ω will be extracted from the measurement, the influence of F_y can be neglected. Therefore we can omit F_y by setting it to zero.

In detail we have

$$\begin{aligned} F_z &= T_1 \sin(\omega t + \phi) + T_2 \sin(\omega t + \phi + \varphi) + T_3 \sin(\omega t + \phi + 2\varphi) \\ &\quad + \omega^2 m r \sin(\omega t + \phi + \phi_m), \\ M_x &= F_1 l_1 \sin(\omega t + \phi) + F_2 l_2 \sin(\omega t + \phi + \varphi) + F_3 l_3 \sin(\omega t + \phi + 2\varphi) + L F_z, \\ M_z &= F_1 l_1 \cos(\omega t + \phi) + F_2 l_2 \cos(\omega t + \phi + \varphi) + F_3 l_3 \cos(\omega t + \phi + 2\varphi) \\ &\quad + L [T(x_1) \cos(\omega t + \phi) + T(x_2) \cos(\omega t + \phi + \varphi) + T(x_3) \cos(\omega t + \phi + 2\varphi) \\ &\quad + \omega^2 m r \cos(\omega t + \phi + \phi_m)]. \end{aligned}$$

We use the trigonometric identities to decompose F_z , M_x , and M_z into cosine and sine parts of the argument ωt :

$$\begin{aligned} F_z &= F_z^c \cos(\omega t) + F_z^s \sin(\omega t), \\ M_x &= M_x^c \cos(\omega t) + M_x^s \sin(\omega t), \\ M_z &= M_z^c \cos(\omega t) + M_z^s \sin(\omega t). \end{aligned}$$

With the definitions

$$\begin{aligned} c_\phi &:= \cos(\phi), & c_1 &:= \cos(\varphi) = \cos(2\pi/3) = \cos(4\pi/3) = \cos(2\varphi) = -0.5, \\ s_\phi &:= \sin(\phi), & s_1 &:= \sin(\varphi) = \sin(2\pi/3) = -\sin(4\pi/3) = -\sin(2\varphi) \approx 0.866, \end{aligned}$$

we get

$$\begin{aligned} F_z^c &= T_1 s_\phi + T_2 [s_\phi c_1 + c_\phi s_1] + T_3 [s_\phi c_1 - c_\phi s_1] + \omega^2 [s_\phi x_4 + c_\phi x_5], \\ F_z^s &= T_1 c_\phi + T_2 [c_\phi c_1 - s_\phi s_1] + T_3 [c_\phi c_1 + s_\phi s_1] + \omega^2 [c_\phi x_4 - s_\phi x_5], \\ M_x^c &= F_1 l_1 s_\phi + F_2 l_2 [s_\phi c_1 + c_\phi s_1] + F_3 l_3 [s_\phi c_1 - c_\phi s_1] + L F_z^c, \\ M_x^s &= F_1 l_1 c_\phi + F_2 l_2 [c_\phi c_1 - s_\phi s_1] + F_3 l_3 [c_\phi c_1 + s_\phi s_1] + L F_z^s, \\ M_z^c &= (F_1 l_1 + L T_1) c_\phi + (F_2 l_2 + L T_2) [c_\phi c_1 - s_\phi s_1] + (F_3 l_3 + L T_3) [c_\phi c_1 + s_\phi s_1] \\ &\quad + L \omega^2 [c_\phi x_4 - s_\phi x_5], \\ M_z^s &= -(F_1 l_1 + L T_1) s_\phi - (F_2 l_2 + L T_2) [s_\phi c_1 + c_\phi s_1] - (F_3 l_3 + L T_3) [s_\phi c_1 - c_\phi s_1] \\ &\quad - L \omega^2 [s_\phi x_4 + c_\phi x_5]. \end{aligned} \tag{10}$$

According to our model and the corresponding nodes the load vector from aerodynamic and mass imbalances is given by

$$\begin{aligned} \mathbf{p}(t) &= \mathbf{p}_c \cos(\omega t) + \mathbf{p}_s \sin(\omega t) \quad \text{with} \\ \mathbf{p}_c &:= (0, \dots, 0, F_z^c, M_x^c, 0, M_z^c)^T, \\ \mathbf{p}_s &:= (0, \dots, 0, F_z^s, M_x^s, 0, M_z^s)^T. \end{aligned} \tag{11}$$

3.3 Derivation of the forward operator \mathbf{A}

3.3.1 The imbalance vector \mathbf{x} and the data vector \mathbf{y}

The imbalance vector \mathbf{x} contains the pitch angle deviations as well as the information on mass imbalances. It is given by

$$\mathbf{x} = (x_1, x_2, x_3, x_4, x_5) = (\theta_1, \theta_2, \theta_3, mr \cos(\phi_m), mr \sin(\phi_m)), \quad (12)$$

where the θ_i are the pitch angle deviations of each blade, and the mass imbalance is split into real part $x_4 = \Re(mr \exp(i\phi_m))$ and imaginary part $x_5 = \Im(mr \exp(i\phi_m))$. The forward operator \mathbf{A} maps the imbalance \mathbf{x} onto the data \mathbf{y} . We obtain the data by measuring the amplitude c and phase shift γ of the vibration with frequency ω at each sensor position. We have at most 3 sensors located at the last (upper) node of the model. The corresponding DOF at this node have the numbers $\{N-4, N-3, N-2\}$. We can measure

1. the vibration $c_y(\mathbf{x}) \cos(\omega t - \gamma_y(\mathbf{x}))$ at DOF $(N-4)$ (displacement in y -direction),
2. the vibration $c_z(\mathbf{x}) \cos(\omega t - \gamma_z(\mathbf{x}))$ at DOF $(N-3)$ (displacement in z -direction),
3. the vibration $c_{\beta_x}(\mathbf{x}) \cos(\omega t - \gamma_{\beta_x}(\mathbf{x}))$ at DOF $(N-2)$ (torsion around the x -axis).

For all vibrations we record the cosine and sine part of the vibration, cf. (13). Using

$$c \cos(\omega t - \gamma) = c \cos(\gamma) \cos(\omega t) + c \sin(\gamma) \sin(\omega t), \quad (13)$$

we can arrange the data vector as

$$\mathbf{y} = \begin{pmatrix} c_y(\mathbf{x}) \cos(\gamma_y(\mathbf{x})) \\ c_z(\mathbf{x}) \cos(\gamma_z(\mathbf{x})) \\ c_{\beta_x}(\mathbf{x}) \cos(\gamma_{\beta_x}(\mathbf{x})) \\ c_y(\mathbf{x}) \sin(\gamma_y(\mathbf{x})) \\ c_z(\mathbf{x}) \sin(\gamma_z(\mathbf{x})) \\ c_{\beta_x}(\mathbf{x}) \sin(\gamma_{\beta_x}(\mathbf{x})) \end{pmatrix}$$

for all three sensors.

3.3.2 Solution of the ODE system

In order to describe the operator \mathbf{A} that maps the vector of imbalance \mathbf{x} onto the data \mathbf{y} we solve (1) for the cosine and the sine part of \mathbf{p} , cf. (11), separately. The solution \mathbf{u} is the sum of the solutions \mathbf{u}_c and \mathbf{u}_s of the two inhomogeneous systems

$$\mathbf{M}\ddot{\mathbf{u}}_c(t) + \mathbf{S}\mathbf{u}_c(t) = \mathbf{p}_c(\mathbf{x}) \cos(\omega t),$$

$$\mathbf{M}\ddot{\mathbf{u}}_s(t) + \mathbf{S}\mathbf{u}_s(t) = \mathbf{p}_s(\mathbf{x}) \sin(\omega t).$$

Although we have neglected the damping in the model of the WT, there will always be some small damping in every system. Therefore the homogeneous solution is decaying. As the vibration data are obtained over a long time period, we can assume that the homogeneous solution has already died away. With the abbreviation

$$\mathbf{B} := (-\omega^2 \mathbf{M} + \mathbf{S})^{-1}$$

the solutions are given by

$$\mathbf{u}_c(t) = \mathbf{B}\mathbf{p}_c \cos(\omega t),$$

$$\mathbf{u}_s(t) = \mathbf{B}\mathbf{p}_s \sin(\omega t)$$

with \mathbf{p}_c and \mathbf{p}_s from (11). Since both vectors \mathbf{p}_c and \mathbf{p}_s have only entries at the positions $\{(N-3), (N-2), N\}$ and $\mathbf{u}(t)$ can only be measured at DOF $\{(N-4), (N-3), (N-2)\}$, we restrict \mathbf{B} , \mathbf{p}_c , \mathbf{p}_s , and \mathbf{u} by withdrawing the relevant rows and columns (subscript r , see (15)) and get

$$\mathbf{u}_r(t) = (\mathbf{B}_r \mathbf{p}_{c,r}) \cos(\omega t) + (\mathbf{B}_r \mathbf{p}_{s,r}) \sin(\omega t).$$

It follows that \mathbf{A} is given by

$$\mathbf{A}(\mathbf{x}) = \mathbf{y} = \begin{pmatrix} \mathbf{B}_r \mathbf{p}_{c,r} \\ \mathbf{B}_r \mathbf{p}_{s,r} \end{pmatrix}, \quad (14)$$

where

$$\mathbf{p}_{c,r}(\mathbf{x}) := \begin{pmatrix} F_z^c \\ M_x^c \\ M_z^c \end{pmatrix}, \quad \mathbf{p}_{s,r}(\mathbf{x}) := \begin{pmatrix} F_z^s \\ M_x^s \\ M_z^s \end{pmatrix}, \quad \text{and} \quad \mathbf{B}_r := \mathbf{B}_{\{[N-4:N-2];[N-3,N-2,N]\}}. \quad (15)$$

3.4 Derivative

We recall that parts of $\mathbf{p}_{c,r}$ and $\mathbf{p}_{s,r}$ occurring in the operator \mathbf{A} are derived with the non-linear BEM method. Hence \mathbf{A} is a nonlinear operator. The Frechet derivative \mathbf{A}' of \mathbf{A} is given by the Jacobi matrix

$$\mathbf{A}'(\mathbf{x}) = \begin{pmatrix} \mathbf{B}_r \\ \mathbf{B}_r \end{pmatrix} \begin{pmatrix} \frac{\partial \mathbf{p}_{c,r}}{\partial x_1} & \dots & \frac{\partial \mathbf{p}_{c,r}}{\partial x_5} \\ \frac{\partial \mathbf{p}_{s,r}}{\partial x_1} & \dots & \frac{\partial \mathbf{p}_{s,r}}{\partial x_5} \end{pmatrix}, \quad (16)$$

with

$$\frac{\partial \mathbf{p}_{c,r}}{\partial x_i} = \begin{pmatrix} \frac{\partial F_z^c}{\partial x_i} \\ \frac{\partial M_x^c}{\partial x_i} \\ \frac{\partial M_z^c}{\partial x_i} \end{pmatrix}, \quad \frac{\partial \mathbf{p}_{s,r}}{\partial x_i} = \begin{pmatrix} \frac{\partial F_z^s}{\partial x_i} \\ \frac{\partial M_x^s}{\partial x_i} \\ \frac{\partial M_z^s}{\partial x_i} \end{pmatrix}.$$

We denote by ∂T_i the partial derivative $\partial T_i / \partial x_i$ and by $\partial (Fl)_i$ the partial derivative $\partial (F_i \cdot l_i) / \partial x_i$. Starting from (10) we have, e.g.

$$\begin{aligned} \frac{\partial F_z^c}{\partial x_1} &= s_\phi \partial T_1, & \frac{\partial F_z^c}{\partial x_2} &= (s_\phi c_1 + c_\phi s_1) \partial T_2, & \frac{\partial F_z^c}{\partial x_3} &= (s_\phi c_1 - c_\phi s_1) \partial T_3, \\ \frac{\partial F_z^c}{\partial x_4} &= \omega^2 s_\phi, & \frac{\partial F_z^c}{\partial x_5} &= \omega^2 c_\phi, & \frac{\partial M_x^c}{\partial x_1} &= s_\phi (\partial (Fl)_1 + L \partial T_1), \\ \frac{\partial M_x^c}{\partial x_2} &= (s_\phi c_1 + c_\phi s_1) (\partial (Fl)_2 + L \partial T_2), & & & \dots \end{aligned}$$

It remains to compute the partial derivatives ∂T_i and $\partial(FI)_i$, $i = 1, 2, 3$. Since T , F , and I are obtained via a nonlinear numerical routine we cannot compute the partial derivatives explicitly. Instead, we use the differential quotients as an approximation provided Δ is small, e.g.

$$\begin{aligned}\frac{\partial T_j}{\partial x_j}(x_j) &\approx \frac{T_j(x_j + \Delta) - T_j(x_j)}{\Delta}, \quad \dots, \\ \frac{\partial(FI)_j}{\partial x_j}(x_j) &\approx \frac{\partial F_j}{\partial x_j}(x_j)l_j + F_j \frac{\partial l_j}{\partial x_j}(x_j), \quad j = 1, 2, 3.\end{aligned}$$

Numerical experiments show, that $\Delta \approx 0.1$ provides a sufficiently accurate approximation of the partial derivatives. Thus, the matrix

$$\partial \mathbf{p} = \begin{pmatrix} \frac{\partial \mathbf{p}_{c,r}}{\partial x_1} & \dots & \frac{\partial \mathbf{p}_{c,r}}{\partial x_5} \\ \frac{\partial \mathbf{p}_{s,r}}{\partial x_1} & \dots & \frac{\partial \mathbf{p}_{s,r}}{\partial x_5} \end{pmatrix} \quad (17)$$

can be computed.

4 Regularization

The inverse problem of (14) is to find the imbalance vector \mathbf{x} for given data \mathbf{y} that, in case of real measurements, are contaminated by noise. We assume that the noisy data \mathbf{y}^δ fulfill the inequality $\|\mathbf{y} - \mathbf{y}^\delta\| \leq \delta$, i.e., a bound on the noise is known. The operator \mathbf{A} is nonlinear and ill-posed. To obtain a stable solution, one has to use regularization methods, [6, 10]. Theoretical results for the regularization of nonlinear problems were derived in the last decade, cf. [11–16]. We have used the most prominent Tikhonov regularization.

4.1 Tikhonov functional

In Tikhonov regularization we determine an approximation \mathbf{x}_α^δ of the solution \mathbf{x} as a minimizer of the Tikhonov functional $J_\alpha(\mathbf{x})$ with a rule to choose the regularization parameter α . We use Morozov's discrepancy principle, i.e., we define a sequence of α_k , e.g., $\alpha_{k+1} = q\alpha_k$, $q < 1$, and compute $\mathbf{x}_{\alpha_k}^\delta$ as minimizer of $J_{\alpha_k}(\mathbf{x})$ for each k and stop the iteration if for the first time

$$\|\mathbf{A}(\mathbf{x}_{\alpha_k}^\delta) - \mathbf{y}^\delta\| \leq \tau\delta, \quad \tau > 1 \text{ fixed},$$

holds. In order to minimize the Tikhonov functional for each α_k we will use a gradient-based method. The Tikhonov functional is given by

$$J_\alpha(\mathbf{x}) = \|\mathbf{A}(\mathbf{x}) - \mathbf{y}^\delta\|^2 + \alpha \|\mathbf{x} - \bar{\mathbf{x}}\|^2,$$

where $\|\cdot\|$ is the usual $l_2(n)$ -norm with $n = 5$, and $\bar{\mathbf{x}}$ is an a priori guess of the solution. In our case, $\bar{\mathbf{x}}$ can be the vector $(0, 0, 0, \overline{mr} \cos(\bar{\varphi}_m), \overline{mr} \sin(\bar{\varphi}_m))$ where the mass imbalance is computed using MIA. In view of the different absolute values of the entries in the vector \mathbf{x} it is appropriate to choose a weighted norm instead of $\|\mathbf{x} - \bar{\mathbf{x}}\|^2$. According to the common

applications we chose $\mathbf{c} = (1, 1, 1, 0.01, 0.01)$ and define

$$\|\mathbf{x}\|_c^2 := \sum_{i=1}^5 c_i x_i^2 = (\mathbf{c} * \mathbf{x})^T \mathbf{x},$$

$$\langle \mathbf{x}, \mathbf{z} \rangle_c := \sum_{i=1}^5 c_i x_i z_i = (\mathbf{c} * \mathbf{x})^T \mathbf{z},$$

where $\mathbf{c} * \mathbf{x} = (c_1 x_1, \dots, c_5 x_5)^T$ denotes the component-by-component multiplication. We remark that $\langle \mathbf{x}, \mathbf{z} \rangle_c = \langle \mathbf{c} * \mathbf{x}, \mathbf{z} \rangle$.

Hence the Tikhonov functional now reads

$$\begin{aligned} J_\alpha(\mathbf{x}) &= \|\mathbf{A}(\mathbf{x}) - \mathbf{y}^\delta\|^2 + \alpha \|\mathbf{x} - \bar{\mathbf{x}}\|_c^2 \\ &= \|\mathbf{A}(\mathbf{x}) - \mathbf{y}^\delta\|^2 + \alpha \|\mathbf{c} * (\mathbf{x} - \bar{\mathbf{x}})\|^2. \end{aligned} \quad (18)$$

4.2 The gradient of Tikhonov functional

In order to minimize the functional, the gradient has to be computed. Since \mathbf{A} is nonlinear we use the Taylor expansion of $\mathbf{A}(\mathbf{x} + \mathbf{h})$,

$$\mathbf{A}(\mathbf{x} + \mathbf{h}) = \mathbf{A}(\mathbf{x}) + \mathbf{A}'(\mathbf{x})\mathbf{h} + O(\mathbf{h}^2),$$

where \mathbf{A}' is the Frechet derivative of \mathbf{A} computed in (16) and (17). We have

$$\begin{aligned} J_\alpha(\mathbf{x} + \mathbf{h}) &= \|\mathbf{A}(\mathbf{x}) - \mathbf{y}^\delta\|^2 + \alpha \|\mathbf{x} - \bar{\mathbf{x}}\|_c^2 \\ &\quad + 2\langle \mathbf{c} * \{[\mathbf{A}'(\mathbf{x})]^T (\mathbf{A}(\mathbf{x}) - \mathbf{y}^\delta)\} + \alpha(\mathbf{x} - \bar{\mathbf{x}}), \mathbf{h} \rangle_c + O(\mathbf{h}^2) \\ &= J_\alpha(\mathbf{x}) + 2\langle [\mathbf{A}'(\mathbf{x})]^T (\mathbf{A}(\mathbf{x}) - \mathbf{y}^\delta) + \alpha(\mathbf{c} * (\mathbf{x} - \bar{\mathbf{x}})), \mathbf{h} \rangle + O(\mathbf{h}^2). \end{aligned}$$

It follows that

$$\begin{aligned} \frac{1}{2} \nabla J_\alpha(\mathbf{x}) &= \mathbf{c} * \{[\mathbf{A}'(\mathbf{x})]^T (\mathbf{A}(\mathbf{x}) - \mathbf{y}^\delta)\} + \alpha(\mathbf{x} - \bar{\mathbf{x}}) \\ &= [\mathbf{A}'(\mathbf{x})]^T (\mathbf{A}(\mathbf{x}) - \mathbf{y}^\delta) + \alpha \mathbf{c} * (\mathbf{x} - \bar{\mathbf{x}}). \end{aligned}$$

4.3 Minimization with steepest decent and algorithm

The functional (18) can be minimized for each fixed α by the steepest decent method. Starting from an initial value \mathbf{x}_0 the minimizer is computed iteratively by updating the current iterate in the direction of steepest descent, i.e., the direction of the negative gradient:

$$\begin{aligned} \mathbf{x}_{k+1} &= \mathbf{x}_k - \beta \nabla J_\alpha(\mathbf{x}) \\ &= \mathbf{x}_k + \beta 2\{[\mathbf{A}'(\mathbf{x}_k)]^* (\mathbf{y}^\delta - \mathbf{A}(\mathbf{x}_k)) - \alpha \mathbf{c} * (\mathbf{x}_k - \bar{\mathbf{x}})\}. \end{aligned}$$

There are theoretical results on how to choose the step size β , [6, 13]. In the numerical realization, we have chosen $\beta = 1/(2\alpha^2)$. If α becomes very small, β is chosen fixed, e.g. $\beta = 10$. The final minimization algorithm is shown in Algorithm 4.

Algorithm 4

CHOOSE

$\alpha, q < 1, \tau > 1, x_0$, estimate δ

WHILE $\|A(x) - y^\delta\| > \tau \delta$

$\alpha = q\alpha, \beta = \frac{1}{2\alpha^2}$

if $\beta > 10$ **then** $\beta = 10$ **end**

WHILE $\frac{|J_\alpha(x_0) - J_\alpha(x_1)|}{|J_\alpha(x_0)|} > \varepsilon$

 Compute $A'(x_0)$

$x_1 = x_0 + \beta 2\{[A'(x_0)]^*(y^\delta - A(x_0)) - \alpha c \cdot (x_0 - \bar{x})\}$

$x_0 = x_1$

end

end

5 Numerical tests

5.1 Settings

5.1.1 Model data

We have tested the imbalance reconstruction algorithm for a wind turbine of the type NTK 500 with constant rotational frequency, since the aerodynamic data for this WT can be found in the literature. Namely, we use the following system parameters:

- Rotational speed $\Omega_{\text{rot}} = 0.45 \text{ Hz} = 27 \text{ rpm}$.
- Aerodynamic data: NACA63-421 (lift and drag coefficients: <http://airfoiltools.com/airfoil/details?airfoil=naca634421-il>), NKT500-41 (profile data: <http://www.readbag.com/130-226-17-201-extra-web-docs-nordtank-wt-description>), blade length = 20.5 m, wind speed = 8 m/s.
- Model data:
 - Masses: $m_r = 9,030 \text{ kg}$ (rotor), $m_n = 15,400 \text{ kg}$ (nacelle), $m_t = 22,500 \text{ kg}$ (tower).
 - Geometry: Tower height: $h = 33.8 \text{ m}$, outer diameter at tower top: $d_{\text{out}}^{\text{top}} = 1.69 \text{ m}$, outer diameter at tower bottom: $d_{\text{out}}^{\text{bot}} = 2.4 \text{ m}$, distance from rotor to nacelle midpoint $L = 1.5 \text{ m}$.
 - Eigenfrequency: $\Omega_0 = 0.925 \text{ Hz}$.

We have set the zero mark at blade A. Thus blade B correspond to an angle of 240° and blade C corresponds to 120° , resp. For example, assuming a pitch angle deviation of blade B of 2° compared to blade A and C and a mass imbalance of 50 kgm at blade A, the corresponding imbalance vector is $x = (0, 2, 0, 50, 0)^T$; if the imbalance is at blade C, we have $x = (0, 2, 0, 50 \cos(2/3\pi), 50 \sin(2/3\pi))^T$. We obtain radial, axial and torsion vibration data y by applying the forward operator to x . To simulate real measurements, the data y is contaminated with Gaussian noise. The disturbed data y^δ as well as the noise $\delta = \|y - y^\delta\|$ are used as input data for the reconstruction process.

5.1.2 Reconstruction quality

Due to the different magnitudes of mass imbalance and pitch angle deviation we have to choose an appropriate metric in order to evaluate the reconstruction quality of our method. Additionally, we observe that classes of pitch angle vectors actually represent the same pitch angle deviation, namely

$$(x_1, x_2, x_3) \sim (x_1 + s, x_2 + s, x_3 + s), \quad s \in \mathbb{R}.$$

In order to obtain accuracy results we chose from the class of the reconstructed pitch angle deviation the representative $(x_{\alpha,1}^\delta + s, x_{\alpha,2}^\delta + s, x_{\alpha,3}^\delta + s)$ with the smallest distance to the original pitch angle deviation (x_1, x_2, x_3) , i.e. we chose s s.t.

$$s = \arg \min_{t \in \mathbb{R}} \|(x_1, x_2, x_3) - (x_{\alpha,1}^\delta + t, x_{\alpha,2}^\delta + t, x_{\alpha,3}^\delta + t)\|_2.$$

As one can easily verify s is given by

$$s = \frac{1}{3}[(x_1 - x_{\alpha,1}^\delta) + (x_2 - x_{\alpha,2}^\delta) + (x_3 - x_{\alpha,3}^\delta)]. \quad (19)$$

In practice the original pitch angle deviation is not known. Here we chose the number s in a way that at least one of the x_i , $i = 1, 2, 3$, is zero. The residual vibrations are the same for all representatives of the class.

We split the norm of the deviation of the reconstruction \mathbf{x}_α^δ and the original imbalance \mathbf{x} into two parts. The first part is given by the pitch angle deviations,

$$\varepsilon_{\text{pitch}}(\mathbf{x}_\alpha^\delta, \mathbf{x}) := \|(x_1, x_2, x_3) - (x_{\alpha,1}^\delta + s, x_{\alpha,2}^\delta + s, x_{\alpha,3}^\delta + s)\|_2,$$

where s is given by (19). The second part is the difference for the mass imbalance in the usual l_2 -norm:

$$\varepsilon_{\text{mass}}(\mathbf{x}_\alpha^\delta, \mathbf{x}) := \|(x_{\alpha,4}^\delta, x_{\alpha,5}^\delta) - (x_4, x_5)\|_2.$$

In this way we can give a sophisticated account of the reconstruction accuracy of both imbalances types separately. The relative errors are defined by

$$\varepsilon_{\text{pitch}}^{\text{rel}}(\mathbf{x}_\alpha^\delta, \mathbf{x}) := \frac{\varepsilon_{\text{pitch}}(\mathbf{x}_\alpha^\delta, \mathbf{x})}{\|(x_{\alpha,1}^\delta + s, x_{\alpha,2}^\delta + s, x_{\alpha,3}^\delta + s)\|_2},$$

$$\varepsilon_{\text{mass}}^{\text{rel}}(\mathbf{x}_\alpha^\delta, \mathbf{x}) := \frac{\varepsilon_{\text{mass}}(\mathbf{x}_\alpha^\delta, \mathbf{x})}{\|(x_{\alpha,4}^\delta, x_{\alpha,5}^\delta)\|_2},$$

provided the denominator is not zero.

5.2 Aerodynamic imbalance caused by pitch angle deviation

The first test was concerned with the reconstruction of pitch angle deviation imbalances in the absence of mass imbalances. Here we present two examples.

Example 1 We have chosen a pitch angle deviation of 2° at blade B, i.e., the imbalance vector is given by $\mathbf{x} = (0, 2, 0, 0, 0)^T$. Figure 5, top picture, shows the original and the reconstructed pitch angle deviations for a relative noise of $\delta_{\text{rel}} = 10\%$. In Table 1 the reconstructed pitch angle deviation as well as the reconstruction error is given for several noise levels. Since the mass imbalance is zero, the absolute mass imbalance error is given. In Figure 6 the reconstruction error as a function of the noise level and the development of the residual $\|\mathbf{A}(\mathbf{x}_n) - \mathbf{y}^\delta\|$ during the iteration process are presented. The original vibrations in the imbalanced case and the residual vibrations after balancing with the reconstructed balancing setting are shown in Figure 7. The original vibrations are relatively small due to

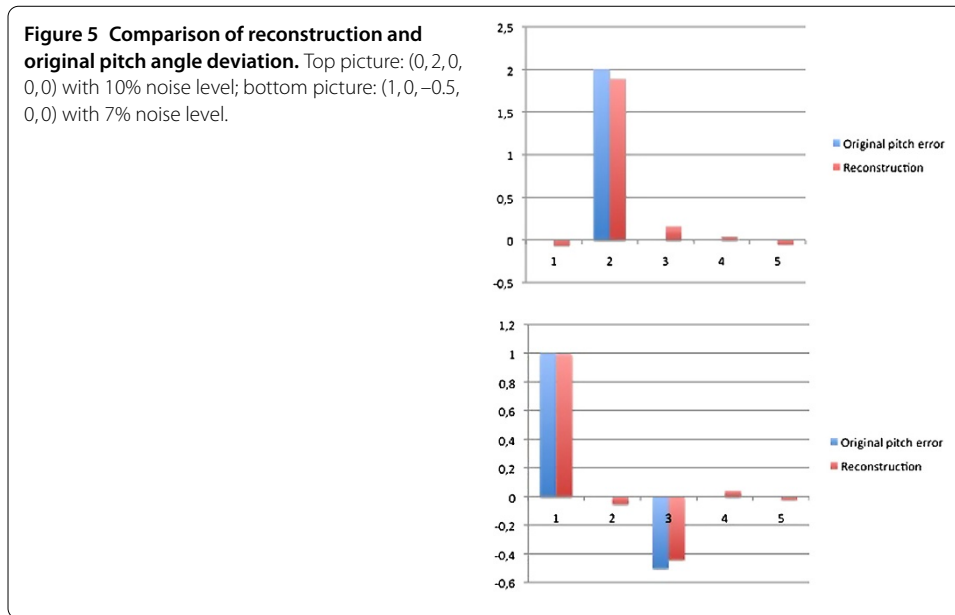


Table 1 Reconstruction results for imbalance $\mathbf{x} = (0, 2, 0, 0, 0)^T$.

δ_{rel}	s	$\mathbf{x}_{\alpha}^{\delta} + s$	$\varepsilon_{\text{pitch}}^{\text{rel}}(\mathbf{x}_{\alpha}^{\delta}, \mathbf{x})$	$\varepsilon_{\text{mass}}(\mathbf{x}_{\alpha}^{\delta}, \mathbf{x})$
3%	0.67	$(-0.03, 1.93, -0.05, 0.04, -0.06)^T$	5.9%	0.07
5%	0.67	$(0.06, 1.97, -0.04, 0.03, -0.05)^T$	4.6%	0.07
8%	0.67	$(0.10, 1.94, -0.04, 0.04, -0.05)^T$	8.5%	0.06
10%	0.67	$(-0.06, 1.89, 0.16, 0.04, -0.05)^T$	13.2%	0.06
12%	0.67	$(-0.06, 1.88, 0.18, 0.02, -0.06)^T$	15%	0.07
15%	0.67	$(0.25, 1.82, -0.07, 0.02, -0.06)^T$	21.5%	0.07
17%	0.67	$(-0.09, 1.83, 0.25, 0.03, -0.06)^T$	21.7%	0.07

the fact that the turbine rotates with a fixed rotor frequency which is only about 70% of the tower eigenfrequency. The remaining vibrations after balancing are almost invisible.

Example 2 The reconstruction result for an imbalance $\mathbf{x} = (1, 0, -0.5, 0, 0)^T$ for a reconstruction with noisy data and noise level of $\delta_{\text{rel}} = 7\%$ is shown in Figure 5, bottom picture. Here, the parameter s is computed as $s = 0.167$ and the relative pitch angle reconstruction error is $\varepsilon_{\text{pitch}}^{\text{rel}}(\mathbf{x}_{\alpha}^{\delta}, \mathbf{x}) = 7\%$.

The initial approximation \mathbf{x}_0 for both examples was chosen as $\mathbf{x}_0 = (0, 0, 0, 0, 0)^T$.

5.3 Combined mass and aerodynamic imbalance

As a second test we have considered a combined imbalance setting, a pitch angle deviation of 2° of blade B and a mass imbalance of 50 kgm at blade A. Blade A corresponds to an initial angle of 0° , hence $\mathbf{x} = (0, 2, 0, 50, 0)^T$. The data were disturbed with a noise level of 7%. To test the performance regarding the initial value for the iteration process we have chosen two different initial approximations: firstly the zero vector $\mathbf{x}_0 = (0, 0, 0, 0, 0)^T$ and secondly $\mathbf{x}_0 = (0, 0, 0, 51.5, -4.8)^T$ that represents zero pitch angle deviation and a mass imbalance of 51.8 kgm at an angle of -5° . This mass imbalance value was reconstructed from the noisy data with MIA. The reconstruction results for both start vectors are presented in Figure 8 and Table 2.

Figure 6 Reconstruction performance.

Reconstruction error versus noise level (top picture), residual during iteration (bottom picture).

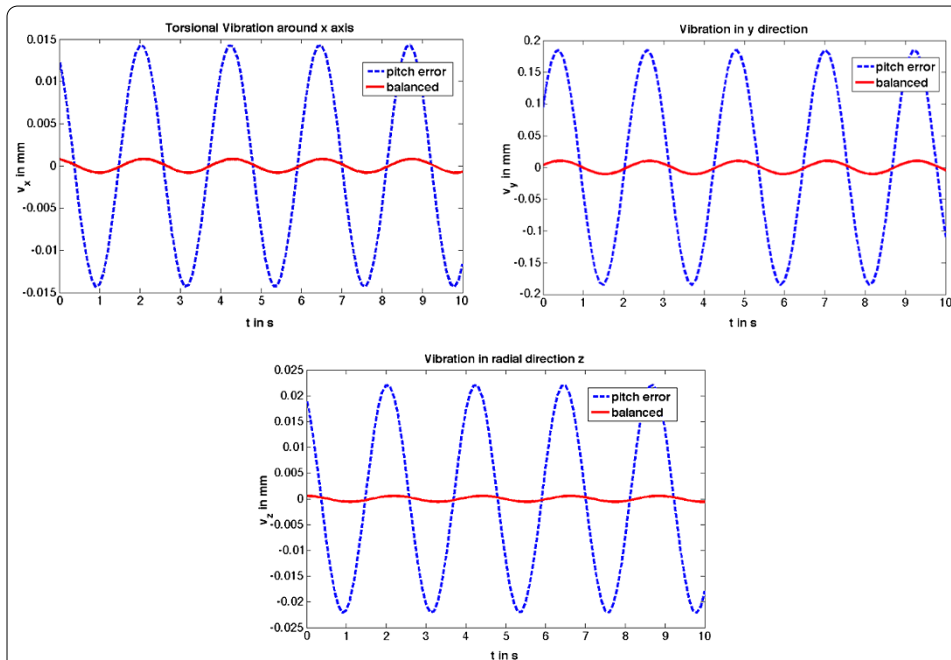
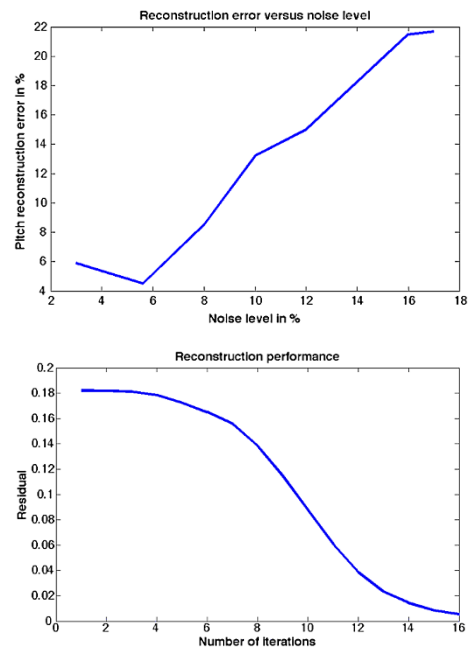


Figure 7 Imbalance vibrations and residual vibrations after balancing.

We observe that the reconstruction process with zero vector as initial imbalance vector converges more slowly but results in a better pitch angle reconstruction. The reconstruction with the precomputed mass imbalance is faster and gives good results for both imbalance types. The original imbalance vibrations and residual vibrations after balancing are plotted in Figure 9 for all vibration directions. The amplitude of the vibrations is reduced

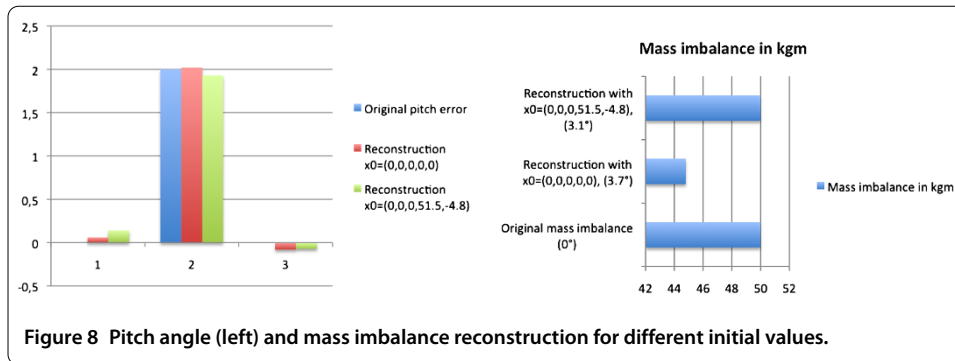
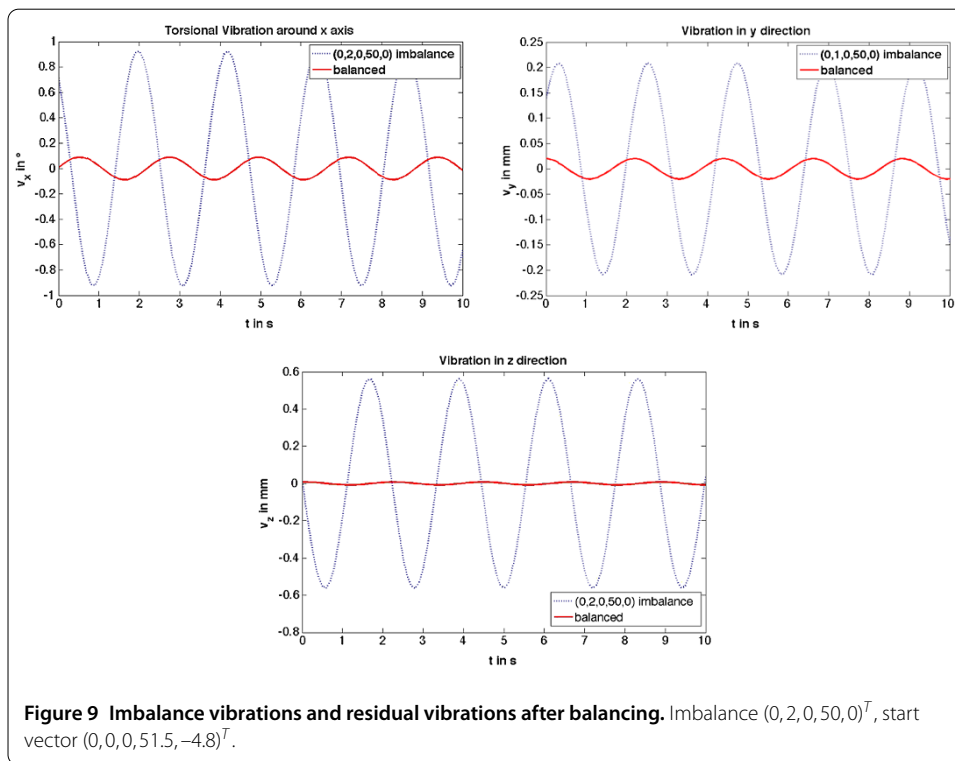


Table 2 Reconstruction results for imbalance $x = (0, 2, 0, 50, 0)^T$.

Start vector	t	$x_{\alpha}^{\delta} + s$	$\varepsilon_{\text{pitch}}^{\text{rel}}(x_{\alpha}^{\delta}, x)$	$\varepsilon_{\text{mass}}^{\text{rel}}(x_{\alpha}^{\delta}, x)$
$(0, 0, 0, 0, 0)^T$	403 s	$(0.06, 2.02, -0.08, 44.7, 2.88)^T$	6%	13%
$(0, 0, 0, 51.5, -4.8)^T$	120 s	$(0.14, 1.93, -0.07, 50.1, 0.67)^T$	11%	1.4%



to less than 10% in case of torsional and axial vibrations and even 2% in case of radial vibration.

As mentioned in the Introduction, the problem of simultaneous reconstruction of mass imbalances and pitch angle deviation was already treated in [5]. In this first attempt, a direct search method for minimizing the nonlinear Tikhonov functional was used, which was far too slow for extensive test computations. Also convergence was not always achieved. With our gradient-based reconstruction method the computation time was re-

duced significantly to a more reasonable time of about 2 minutes on 2.8 GHz Intel Core2 Duo processor. Convergence could also be observed in all tests.

6 Summary and outlook

The paper deals with the mathematical determination of two types of imbalances in a WT: aerodynamic imbalances from pitch angle deviation of the blades and mass imbalances of the rotor. Presently, both types have to be determined on-site by an expert team performing time consuming measurement procedures. The mathematical approach provides a reconstruction method that can be implemented into a condition monitoring system. Instead of the expert team, the method requires the knowledge of geometrical and physical parameters of the WT and aerodynamic airfoil data of the blades. Whereas the former usually can be obtained from inspection reports, the latter represent very sensible data and are hardly available for newer WT. For that reason, so far the method could only be tested for an older NTK500 turbine. Unfortunately, our industrial partners could not provide the necessary vibration data for this type. Thus we had to restrict all tests to artificial data. For future applicability, the WT owners and manufacturers need to be persuaded that it is beneficial for the safe operation of the turbine to provide the sensible airfoil data in a confined way.

The problem of imbalance determination was addressed by the authors in former papers. In this paper we have presented an improved algorithm to reconstruct mass imbalance and pitch angle deviation of the rotor of a WT at the same time. Including pitch angle deviations into the reconstruction leads to a nonlinear forward operator. The regularization of the inverse problem requires the minimization of the Tikhonov functional. Straight forward direct search method proved to be too slow and not stable enough for our purpose. Hence we employed gradient based minimization methods. To that end, we have derived the Frechet derivative of the forward operator that occurs in the gradient of the Tikhonov functional. The accelerated algorithm was successfully applied to several test examples with artificial vibration data. In case of real vibration data we expect equally good reconstruction results as in the test computations, provided that the necessary aerodynamic data of the blades are available.

Competing interests

The authors declare that they have no competing interests.

Authors' contributions

Both authors contributed equally to the writing of this paper. All authors read and approved the final manuscript.

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References

1. Donth A, Grunwald A, Heilmann C, Melsheimer M: **Improving performance of wind turbines through blade angle optimization and rotor balancing.** EWEA O/M strategies 350; 2011.
2. Caselitz P, Giebhardt J: **Rotor condition monitoring for improved operational safety of offshore wind energy converters.** *ASME J Sol Energy Eng* 2005, **127**:253-261.
3. Hameed Z, Hong YS, Cho YM, Ahn SH, Song CK: **Condition monitoring and fault detection of wind turbines and related algorithms: a review.** *Renew Sustain Energy Rev* 2009, **13**:1-39.

4. Ramlau R, Niebsch J: **Imbalance estimation without test masses for wind turbines.** *ASME J Sol Energy Eng* 2009, **131**(1):011010.
5. Niebsch J, Ramlau R, Nguyen TT: **Mass and aerodynamic imbalance estimates of wind turbines.** *Energies* 2010, **3**:696-710.
6. Engl HW, Hanke M, Neubauer A: *Regularization of Inverse Problems*. Dordrecht: Kluwer Academic; 1996.
7. Gasch R, Knothe K: *Strukturdynamik 2*. Berlin: Springer; 1989.
8. Ingram G: **Wind turbine blade analysis using the blade element momentum method**. Durham University, Durham; 2005.
9. Ahlström A: **Aeroelastic simulations of wind turbine dynamics.** *PhD thesis*. Royal Institute of Technology, Department of Mechanics, Stockholm; 2005.
10. Louis AK: *Inverse und Schlecht Gestellte Probleme*. Stuttgart: Teubner; 1989.
11. Kaltenbacher B, Neubauer A, Scherzer O: *Iterative Regularization Methods for Nonlinear Ill-Posed Problems*. Berlin: de Gruyter; 2008.
12. Ramlau R, Teschke G: **Tikhonov replacement functionals for iteratively solving nonlinear operator equations.** *Inverse Probl* 2005, **21**(5):1571-1592.
13. Ramlau R: **TIGRA – an iterative algorithm for regularizing nonlinear ill-posed problems.** *Inverse Probl* 2003, **19**(2):433-4670.
14. Ramlau R: **Morozov's discrepancy principle for Tikhonov regularization of nonlinear operators.** *Numer Funct Anal Optim* 2002, **23**(1,2):147-172.
15. Scherzer O: **The use of Morozov's discrepancy principle for Tikhonov regularization for solving nonlinear ill-posed problems.** *Computing* 1993, **51**:45-60.
16. Ramlau R: **On the use of fixed point iterations for the regularization of nonlinear ill-posed problems.** *J Inverse Ill-Posed Probl* 2005, **13**(2):175-200.

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